

REMOTE MANIPULATION ARM IN TWO PORTIONS

The subject of this invention is a remote manipulation arm comprising a master arm and a slave arm in two separate portions.

The remote manipulation arms consisting of a master arm and of a slave arm, are currently used for working in a hostile medium and comprise mechanical transmissions connecting the joints of the segments of the master arm to those of the slave arm in order to have the movements of the master arm, imposed by an operator, reproduced by the slave arm. The end of the master arm is held by the operator and the end of the slave arm generally comprises a manipulation tool which has to accomplish work.

The remote manipulation arms are generally in a single piece, the master arm being directly connected to the slave arm so that mechanical transmissions are continuous; however there is a design discussed in French Patent 2 667 532, which teaches that the slave arm may be without a master arm and controlled by a device with buttons or of a similar kind, without producing a control movement reproduced by the slave arm: the controls are converted into electric signals provided to a control system which drives the motors which control the transmissions of the slave arm according to these signals. Different, possibly more accurate or more effective laws, for controlling the slave arm may be allowed. Another feature is that the forces in the arm, due to gravity or to the undertaken work, are no longer transmitted to the operator. If the

fatigue of the latter is less, it should be noted that on the contrary it is often sought to have him/her feel an effort proportioned with that experienced by the tool for improving the quality of the control.

5 So-called force feedback devices are added for producing the desired reactions in the master arm. No such reaction may be produced according to this Patent 2 667 532.

The remote manipulation arms generally have 10 limited movements. In this way the pivoting movement of a tubular segment, a so-called crossing segment, engaged through a protective wall and mounted on rolling bearings, is reduced in practice since it is produced by having the master arm pivot upwards, which 15 can only be performed by a small angle, as a too high elevation would no longer allow the operator to suitably hold the master arm and the pivoting of the master arm is accompanied with a same pivoting of the slave arm, the weight of which produces an antagonist 20 moment to the elevation which hinders the operator. Counterweights reduce the pivoting moment but their action remains incomplete in practice. In spite of their possibilities, the slave arms are therefore 25 associated with reduced work space and especially with a low height. The aforementioned patent does not find a remedy to this insufficiency: it contains the indication of having the slave arm pivot by a transversely positioned cylinder which only authorizes a small motion angle.

30 The most important object of the invention is therefore to increase the possibilities of movement in

remote manipulation arms and notably by raising the level where the slave arm may be placed and this without any difficulty for the operator.

5 The layout retained for the new remote manipulator arm implies that the master arm and the slave arm are separated so as to drive the slave arm into large rotations without giving up the benefits of the control by producing a manual movement possibly associated with force feedback to the operator.

10 Under a general definition, the invention relates to a remote manipulator arm comprising a master arm and a slave arm without any direct mechanical transmission linking them together, but provided with an interface system comprising a control portion of a rather 15 electrical and computer-based nature, and a power portion driving the mechanical transmissions included in the slave arm under the control of signals transmitted by the control portion in response to indications of movements accomplished on the master 20 arm; the power portion comprising a stationary casing, a rotatably mounted drum in the casing and attached to the slave arm, and a stationary motor as well as a transmission connecting the drum to the motor and allowing the drum to rotate by one full turn.

25 Keeping the master arm implies keeping the force feedbacks applied to the operator, after converting the forces located in the slave arm into electric signals by the interface system.

These objects of the invention as well as other 30 ones will be now described in connection with the figures:

- Fig. 1 illustrates the general aspect of the invention;

- Fig. 2 illustrates the system for driving the slave arm;

5 - and Figs. 3, 4 and 5 illustrate certain particular aspects of the invention.

Fig. 1 is approached. A slave arm 1 comprises a first tubular segment 3, engaged through a protective wall 2, and a train 4 of other segments jointed with 10 each other and with the first segment 3 and located beyond the protective wall 2 in the hostile work medium; a master arm 5 positioned at a certain distance from the slave arm 3; and an interface system 6 consisting of a control portion 7 and an power portion 15 8 attached to the protective wall 2 and linked to the slave arm 3. Electric lines 9 and 10 connect the control portion 7 to the slave arm 5 and to the power portion 8 for controlling the latter according to the movements of the latter arm. Finally, with a window 11 20 the operator holding the master arm 5 may observe the effect of his/her action on the slave arm 1 and the tool 12 (often a grip).

Fig. 2 is referred to for a main view of the most remarkable part of the invention, i.e., the power 25 portion 8. It basically comprises a stationary casing 13, a drum 14 rotating in the casing 13, a stationary motor 15 for driving 2 the drum 14 and a gear transmission 16 providing this drive, which comprises a pinion on the output shaft of the motor 15 and a crown 30 on the drum 14. The pinion may mesh with the crown or be connected to it through a synchronous belt or

another means. The casing 13 is mounted on the protective wall 2 through an anchoring device 17. The drum 14 is supported by the casing 13 by means of bearings 18 which allow it to rotate, and it extends at 5 least partly in a cylindrical portion 19 of the casing 13. The motor 15 extends in another portion 20 of the casing 13 and is fixed therein. The transmission 16 comprises a pinion 21 attached to the output shaft of the motor 15 and a crown 22 meshing with the pinion 21, 10 surrounding a portion of the drum 14. With this system, it is possible to rotate the drum 14 by a full turn, and more, around an axis coinciding with that of the first segment 3, which is adjacent to the end of the drum 14 and in its extension. The complete link may be 15 established by screwing contacting brackets 23 belonging to both of these parts. The rotation of the drum 14 therefore causes that of the first segment 3 as well as that of the whole slave arm 1.

The drum 14 contains a certain number of control 20 motors, among which a central motor 24 may be distinguished, the other ones (six in number, but only one being illustrated here) being side motors 25. Each of these control motors 24 and 25 is associated with a transmission extending in the slave arm 1 for 25 controlling a segment of the train 4 and which typically comprises a transmission bar in the tubular segment 3. The bar associated with the central control motor 24 bears reference 26, and the bars associated with the other control motors 25 bear reference 27 30 (here, only one is illustrated). The bar 26 is extended into the drum 14 through an output shaft 28 of the

motor 1, and the bars 27, through shafts 29 which are driven by the side motors 25 via synchronous belt transmissions 30 which allow the corresponding side motors 25 to be moved close to the periphery of the 5 drum 14 and they only feature very small play. The bars 26 and 27 are joined to the shafts 28 and 29 by bolt couplings 31 or other couplings. The bar 26 is intended to tilt the segment jointed with the tubular segment 3, and also the remainder of the train 4, in a vertical 10 direction, which may produce a significant reactive moment due to gravity and imposes suitable power to the control motor 24 and suitable resistance of the mechanical transmission, whereas the forces being exerted on the bars 27, the shafts 29 and the side 15 motors 25 are much less significant, so that fragile transmissions 30 may be used. Also, the pivoting of the tubular segment 3 requires that it should withstand a large force, which justifies the use of a resistant gear transmission 16.

20 The electric connectors 31 which receive the wires of the line 10 and which are placed on the casing 13, a cable winder device 32 with which the control motors 24 and 25 may be suitably powered electrically, and an end-of-travel device 33 are further illustrated. The 25 last two devices will now be described with the help of the following figures.

Fig. 3 illustrates the cable winder 32. It is positioned in an annular space between the cylindrical portion 19 containing the drum 14 and the drum 14 30 itself, and it mainly consists in a circular-arc shaped part 34 freely mounted in this space and at least the

ends of which are provided with rollers 35. Further, the cables 36 for connecting to the drum 14 have a point 37 for attachment to the drum 14 and a point 38 for attachment to the casing 13. Their extension is a 5 little more than a circumference of the drum 14. The features of the device of its operation may be summarized in this way: the slave arm 5 and notably the tubular segment 3 should rotate so as to be placed in all the angular positions; displacements of one turn 10 are sufficient for placing the tool 12 in all the possible positions, but a little larger angular travel is required for absorbing the braking of the arm and for triggering the end-of-travel devices. The circular-arc-shaped part 14 distributes the length of 15 the cables 36 in an external strand close to the cylindrical portion 19 and an internal strand close to the drum 14 on either side of one of the rollers 35. This is located here at an end position of the drum 14. By displacing the latter clockwise, the internal strand 20 of the cables 36 is drawn while reducing the length of the external strand and by rotating the circular-arc-shaped part 34. Rotation is allowed until the external strand has disappeared. A suitable position of the cables 36 remains guaranteed by the circular-arc-shaped 25 part 34 which is used as a spacer in any position. The drum 14 may be rotated by one turn and more, without mishap and with little friction. In practice, the cables are placed in a protective chain near its average radius of curvature so as to minimize the 30 friction of the cables on each other.

Fig. 4 illustrates the end-of-travel device 33. A

helix 39 is positioned at the periphery of the drum 14. Two end-of-travel contacts are mounted on the casing 13. A slide 41 is also mounted on the casing 13 and bears a carriage 42 which slides on it. The carriage 42 5 comprises a cam 43 which may come into abutment with the end-of-travel contacts 40 and a pair of rollers 44 encircling the helix 39 and forming a groove of the carriage into which it engages. The rotation of the drum 14 causes the helix 39 to move past between the 10 rollers 44 while displacing them, as well as the remainder of the carriage 42, the slide being parallel to the axis of rotation of the drum 14 and to that of the helix 39. When the cam 43 touches the end-of-travel contacts 40, stopping the movement of the drum 14 is 15 imposed. The end-of-travel contacts 40 are connected to the control portion 7 by electrical lines not shown, for signaling these states. The method provides protection of the winder over its angular rotational travel.

20 A particular enhancement is illustrated by means of Fig. 5. It is applied to certain known slave arms which are characterized in that the last segments of the train 4 are sliding segments in order to vary the length of the slave arm 1, rather than rotating 25 segments. This feature here relates to three end segments 45, 46 and 47. The transmission with which opening out of the middle segment 46 may be adjusted relatively to the preceding segment 45 comprises a worm screw 48 positioned through a tapped bushing 49 30 attached to the middle segment 46, the worm screw 48 being retained in a fixed position of the preceding

segment 45 by flanges 50. It is driven into rotation by gear means with angle transmission, with a transmission bar, etc., passing through the front segments and ending with one of the transmission bars and one of the 5 control motors 25. As the structure of this arm is known, we have only partly illustrated its structure and transmission. The same applies to the transmission for controlling the movement of the end segment 47. We shall merely point out that it comprises a double 10 strand cable component 50, the strands ending up at a pair of pulleys 52 driven by a same control bar 27, respectively. They end up at their opposite end, in a common point of attachment 53 on the end segment 47, one of them further passing through a deflection pulley 15 54 attached to the bottom of the middle segment 46. As the strands are wound on the pulleys 52 in opposite directions, rotation of the bar 27 unwinds one of them and winds the other one, so that they cooperate in opening out and retracting the end segment 47 in the 20 middle segment 46. It should be noted that rotation of the worm screw 48 not only displaces the middle segment 46 but also the end segment 47, since the strands of the cable 51 also pass through another deflection pulley 55 integral with a carriage 56 mounted on the 25 previous segment 45 and which is connected to the middle segment 46 through another cable 57. A displacement of the middle segment 46, downwards, for example slackens the cable 57, which allows the carriage 56 to be raised along the previous segment 45 30 by half displacement, each of the strands of the cable 51 is slackened by the half displacement on either side

of the deflection pulley 55, and the end segment 47 moves down by the same amount as the middle segment 46.

The worm screw transmission 48 is robust, but not that of the cable 51. It happens that the segments of 5 the train 4 have to withstand a significant force, such as a heavy load. Blocking the movement of the cable 51 and only resorting to the movement allowed by the worm screw 48 is then considered. This is achieved by a mechanical brake 58 blocking the rotation of the shaft 10 29 associated with the cable transmission 51 and which may consist in a friction disk or a mobile jaw clutch. The associated motor 25 and transmission 30 are relieved from the force experienced by the control bar 27.

15 A suitable master arm 5 may be the one developed by Haption under reference Virtuose 6D 4040. Its force capacity is limited to 4 kilograms. Its friction (relatively to its capacity) is comparable to that of the slave arm 1 and to that of a conventional remote 20 manipulator. It should also be noted that the control portion 7 may also operate with force feedback, i.e., record the forces experienced by the slave arm 1 by means of the forces which the control motors must provide, or by suitable sensors placed on the motors or 25 the transmissions, and make the operator feel them, by creating forces in motors of the master arm. However, it is provided that the separation of the master arm 5 and of the slave arm 1 will allow the master arm 5 to be controlled more easily, with reduced forces allowing 30 the operator to use both arms together, one in each hand, which will give him much more significant working

possibilities than with the generally tiresome arms known today. Another control mode may be adapted in which a single master arm and an operator may himself/herself control two slave arms of the work 5 station so that the force capacity may in particular be doubled and significant torques may be developed by acting in opposition on a part in two points spaced apart. Finally it is noted that balancing of the slave arm 1 may advantageously be performed accurately by 10 means of the software control which may calculate the torques and forces due to gravity. If need be, balancing by a counterweight will be much easier to achieve in the absence of a master arm which is mechanically connected to it.